



**Corso di laurea Magistrale in Ingegneria Meccanica**

**Tesi di Laurea in  
Meccanica del Veicolo**

**Studio della Sospensione Attiva di un  
autoveicolo**

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**Controrelatore:**

**Chiar.mo Prof. Domenico Laforgia**

**Laureando:**

**Gilberto Delle Rose**



# COSA SI INTENDE PER SOSPENSIONE?

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# ESIGENZE DEL SISTEMA SOSPENSIVO

- COMFORT

$V_s$ .

- GUIDABILITÀ

Accelerazione  
sospesa

della  
massa

Variazione della forza a terra



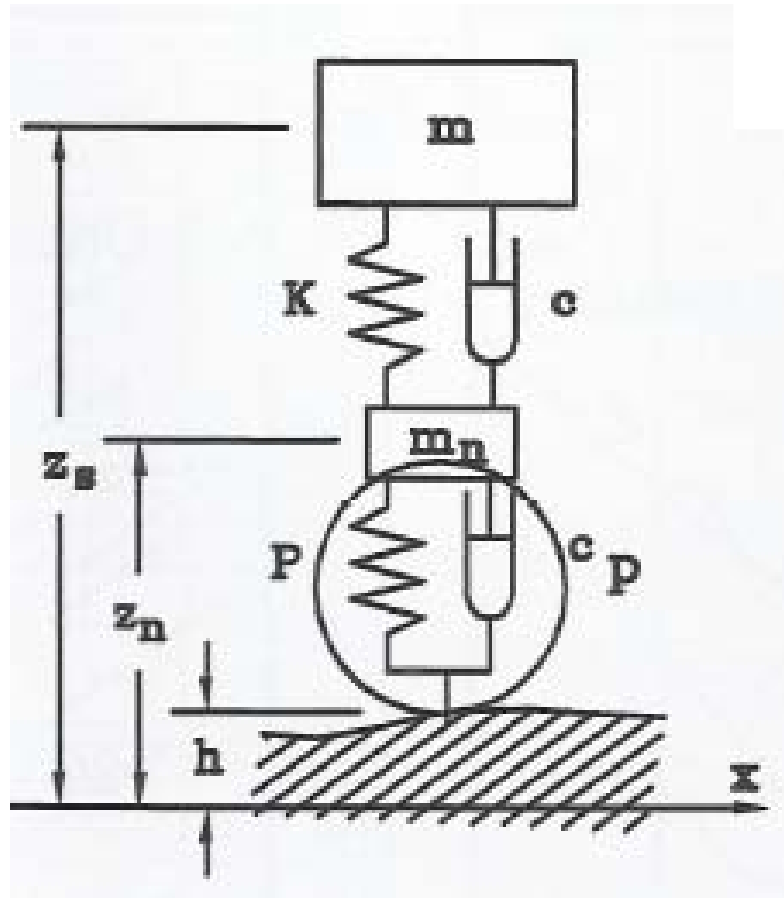
**SOSPENSIONI CONTROLLATE O ADATTIVE**

**OBIETTIVI**



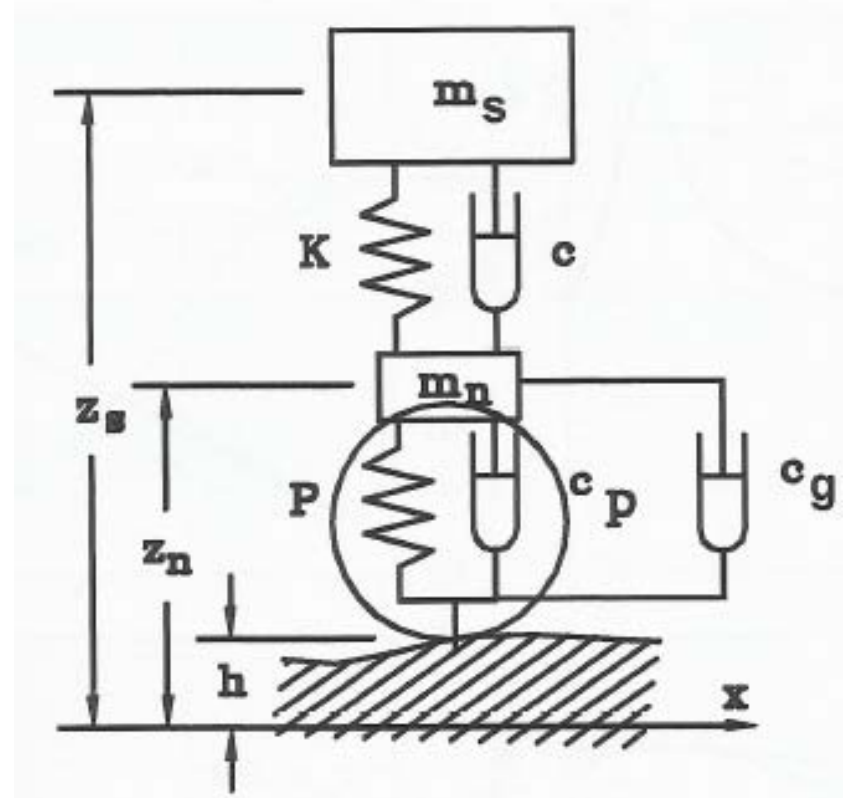
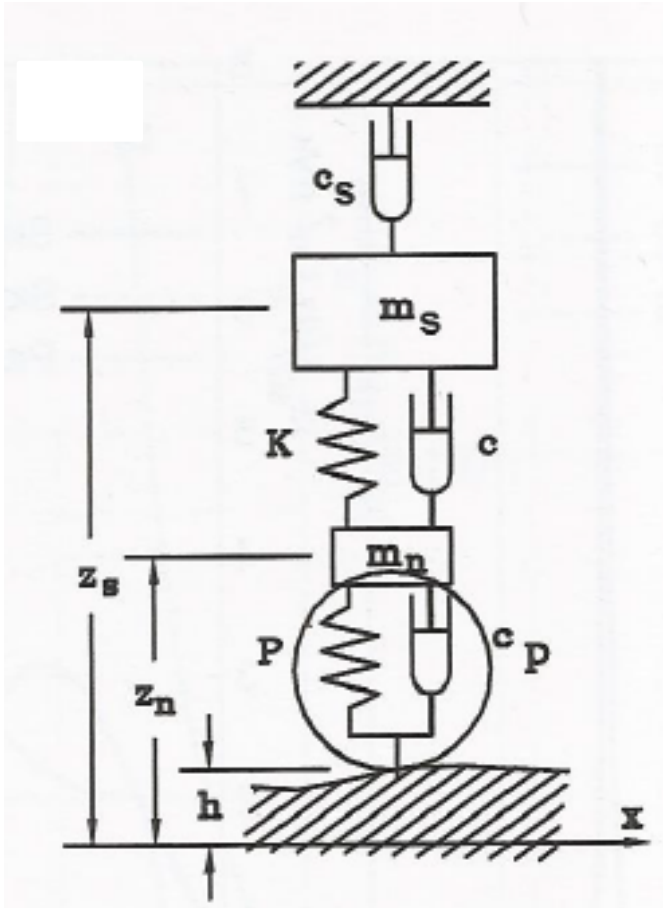
**Realizzare un sistema attivo o inerziale di smorzamento che gestisca queste due variabili contrastanti**

# MODELLO AD $\frac{1}{4}$ DI VEICOLO (2 GDL)



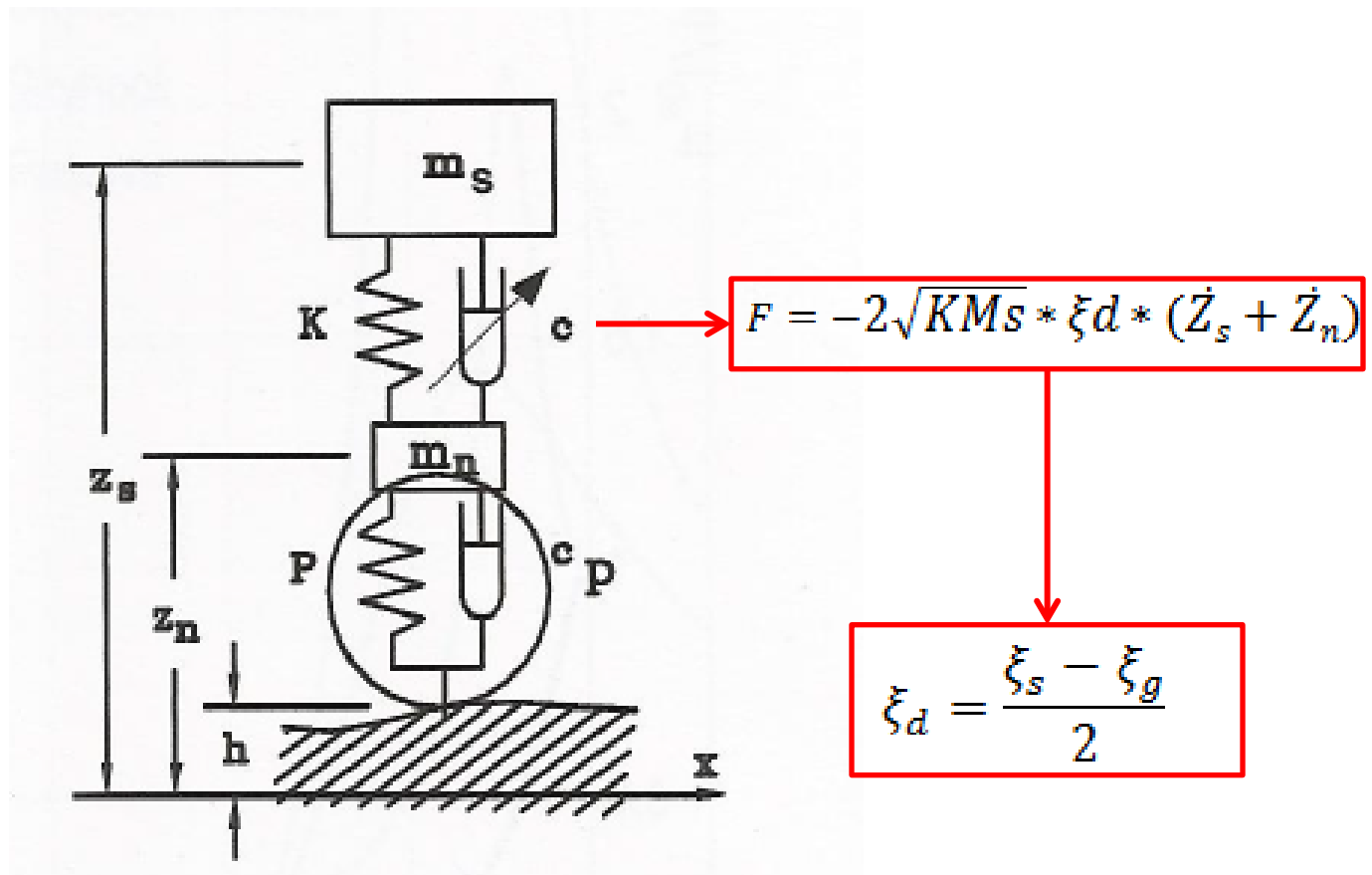


# STRATEGIE DI CONTROLLO SKYHOOK-GROUNDHOOK





# SOSPENSIONE ATTIVA



$$\begin{bmatrix} m_s & 0 \\ 0 & m_n \end{bmatrix} \begin{bmatrix} \ddot{z}_s \\ \ddot{z}_{us} \end{bmatrix} + 2 \begin{bmatrix} \xi_s + \xi & -(\xi_g + \xi) \\ -(\xi_s + \xi) & \xi_g + \xi_s + \xi \end{bmatrix} \sqrt{km_s} \begin{bmatrix} \dot{z}_s \\ \dot{z}_n \end{bmatrix} + \begin{bmatrix} k & -k \\ -k & k \end{bmatrix} \begin{bmatrix} z_s \\ z_n \end{bmatrix} \\
 = \begin{bmatrix} 0 \\ P \end{bmatrix} h(t) + \begin{bmatrix} 0 \\ c_p \end{bmatrix} \dot{h}(t)$$



# MODELLO QUARTER-CAR: FORMA DI STATO

$$\dot{y} = Ay + Bu$$

$$z = Cy + Du$$

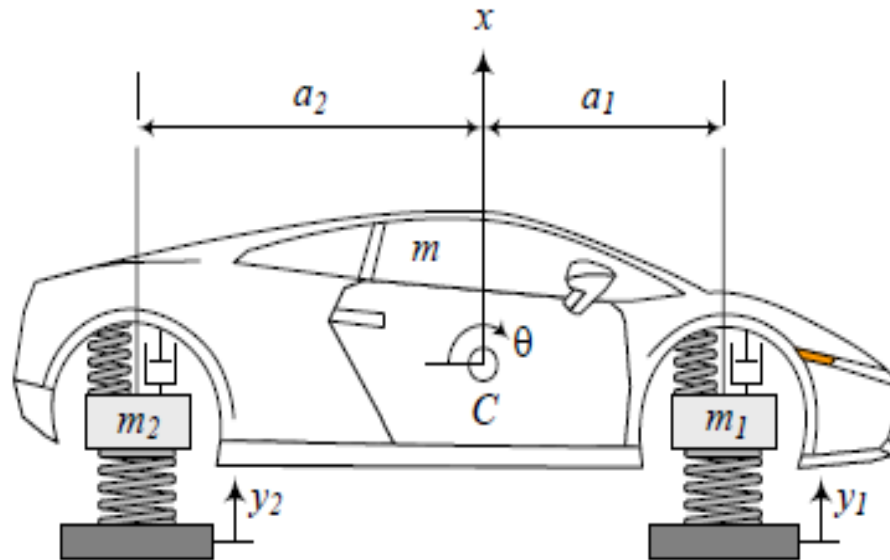
$y_1$  = velocità massa sospesa  
 $y_2$  = spostamento massa sospesa  
 $y_3$  = velocità massa non sospesa  
 $y_4$  = spostamento massa non sospesa

$z_1$  = deformazione pneumatico  
 $z_2$  = corsa sospensione  
 $z_3$  = accelerazione massa sospesa  
 $z_4$  = accelerazione massa non sospesa

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ \frac{k}{m_s} & -\frac{(c_s+c)}{m_s} & \frac{k}{m_s} & \frac{(c_g+c)}{m_s} \\ 0 & 0 & 0 & 1 \\ \frac{k}{m_n} & \frac{(c_s+c)}{m_n} & -\frac{(P+k)}{m_n} & -\frac{(c_g+c_p+c)}{m_n} \end{bmatrix} \quad B = \begin{bmatrix} 0 \\ 0 \\ 0 \\ P/m_n \end{bmatrix}$$

$$C = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 1 & 0 & -1 & 0 \\ \frac{k}{m_s} & -\frac{(c_s+c)}{m_s} & \frac{k}{m_s} & \frac{(c_g+c)}{m_s} \\ \frac{k}{m_n} & \frac{(c_s+c)}{m_n} & -\frac{(P+k)}{m_n} & -\frac{(c_g+c_p+c)}{m_n} \end{bmatrix} \quad D = \begin{bmatrix} -1 \\ 0 \\ 0 \\ P/m_n \end{bmatrix}$$

# MODELLO HALF-CAR (4 gdl)



$$\begin{bmatrix} m_s & 0 & 0 & 0 \\ 0 & I_\theta & 0 & 0 \\ 0 & 0 & m_{n1} & 0 \\ 0 & 0 & 0 & m_{n2} \end{bmatrix} \begin{bmatrix} \ddot{z}_s \\ \ddot{z}_\theta \\ \ddot{z}_{n1} \\ \ddot{z}_{n2} \end{bmatrix}$$

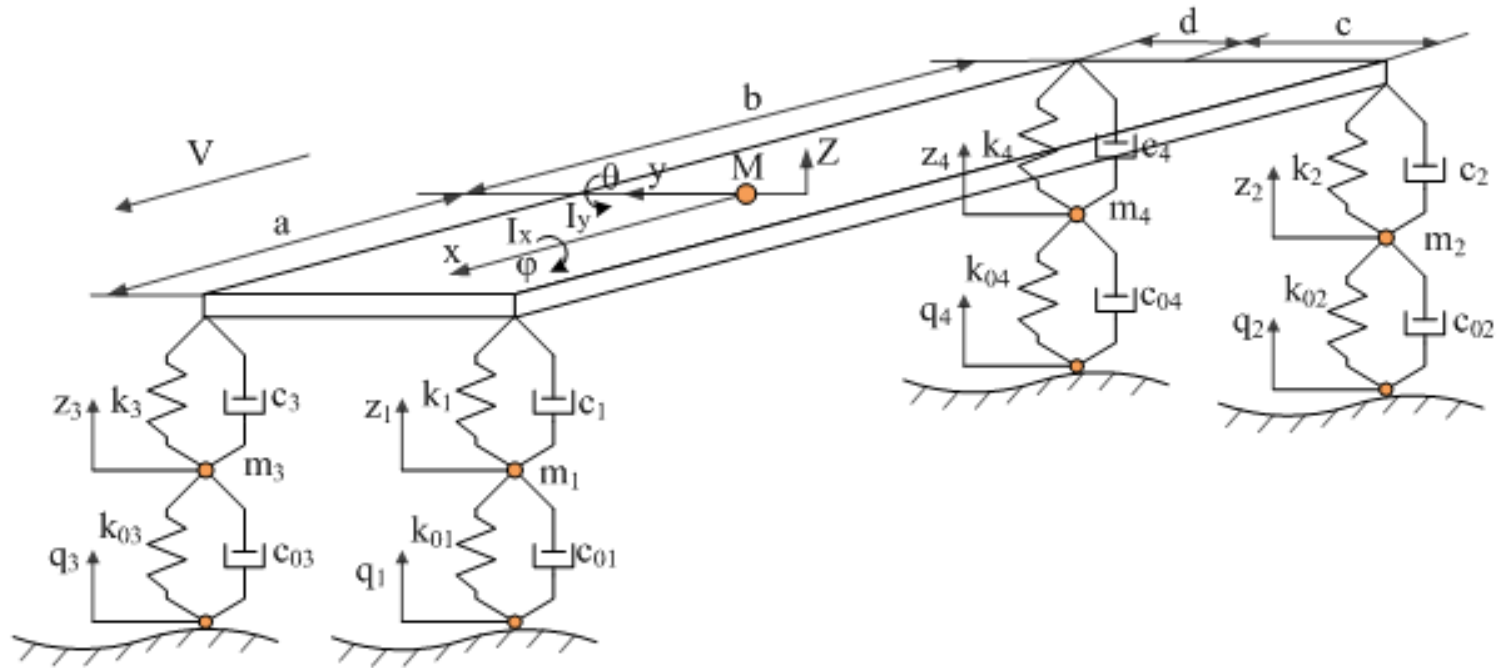
$$+ \begin{bmatrix} c_1 + c_2 - E_1 - E_2 & c_1 b - c_2 a + E_1 a - E_2 b & -c_1 - E_1 & -c_2 - E_2 \\ \text{SYM} & c_1 b^2 + c_2 a^2 + E_1 b^2 + E_2 a^2 & c_1 a + E_1 a & -c_2 b - E_2 b \\ \text{SYM} & \text{SYM} & c_1 + c_{u1} - E_1 & 0 \\ \text{SYM} & \text{SYM} & \text{SYM} & c_2 + c_{u2} - E_2 \end{bmatrix} \begin{bmatrix} \dot{z}_s \\ \dot{z}_\theta \\ \dot{z}_{n1} \\ \dot{z}_{n2} \end{bmatrix}$$

$$+ \begin{bmatrix} k_1 + k_2 & k_1 b - k_2 a & -k_1 & -k_2 \\ \text{SYM} & k_1 b^2 + k_2 a^2 & k_1 a & -k_2 b \\ \text{SYM} & \text{SYM} & k_1 + k_{u1} & 0 \\ \text{SYM} & \text{SYM} & \text{SYM} & k_2 + k_{u2} \end{bmatrix} \begin{bmatrix} z_s \\ z_\theta \\ z_{n1} \\ z_{n2} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ k_{u1} \\ k_{u2} \end{bmatrix} h(t) + \begin{bmatrix} 0 \\ 0 \\ c_{u1} \\ c_{u2} \end{bmatrix} \dot{h}(t)$$



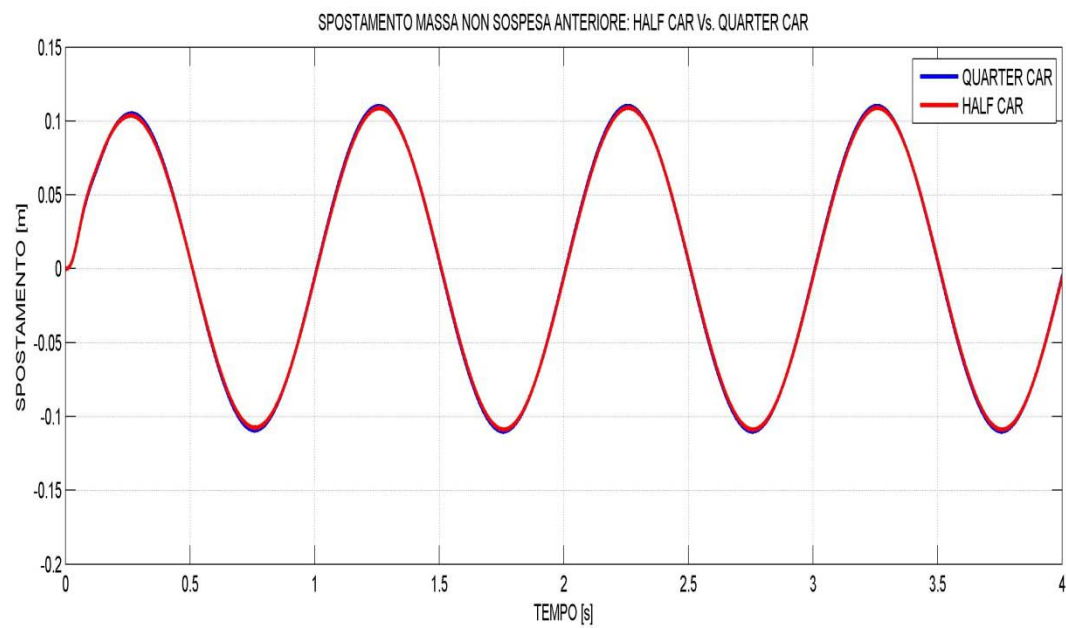
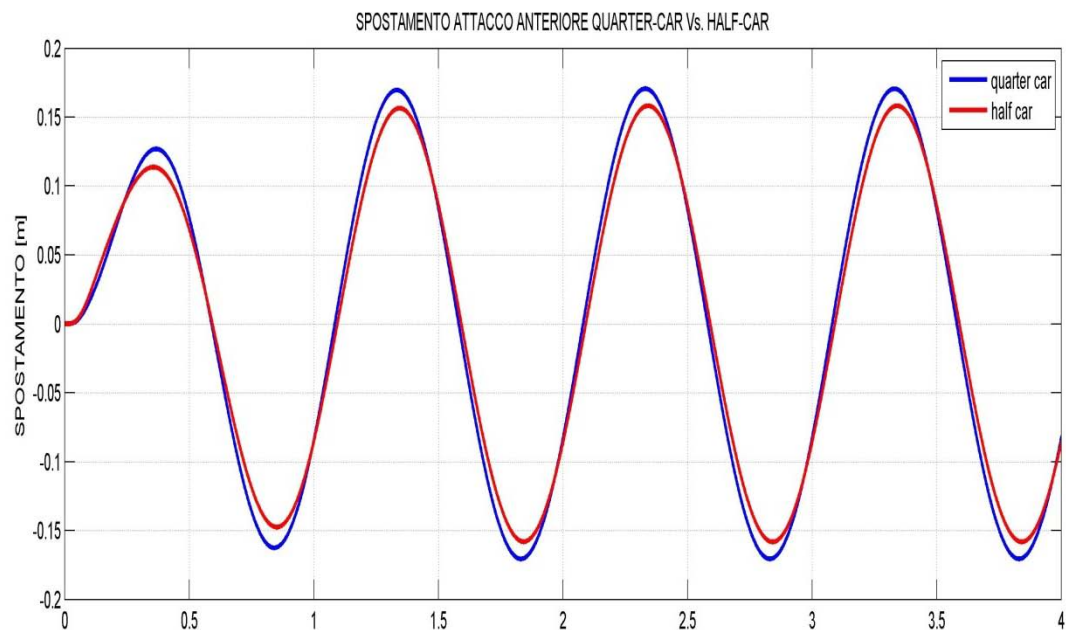


# MODELLO FULL-CAR

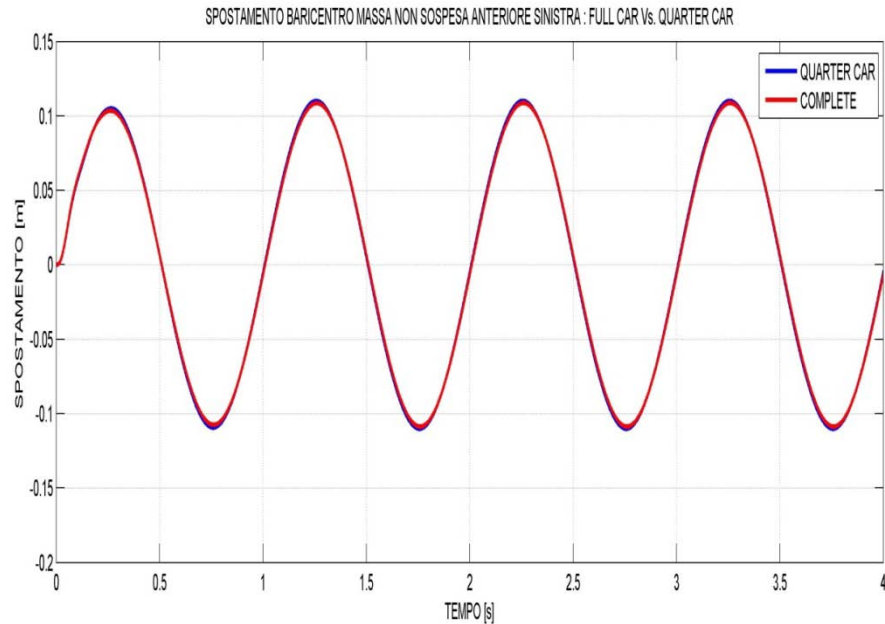
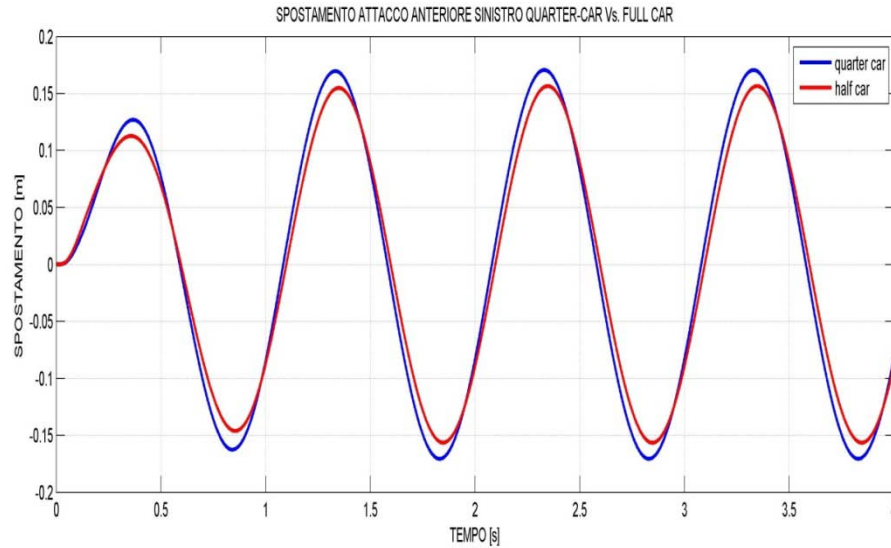




# QUARTER-CAR Vs. HALF-CAR

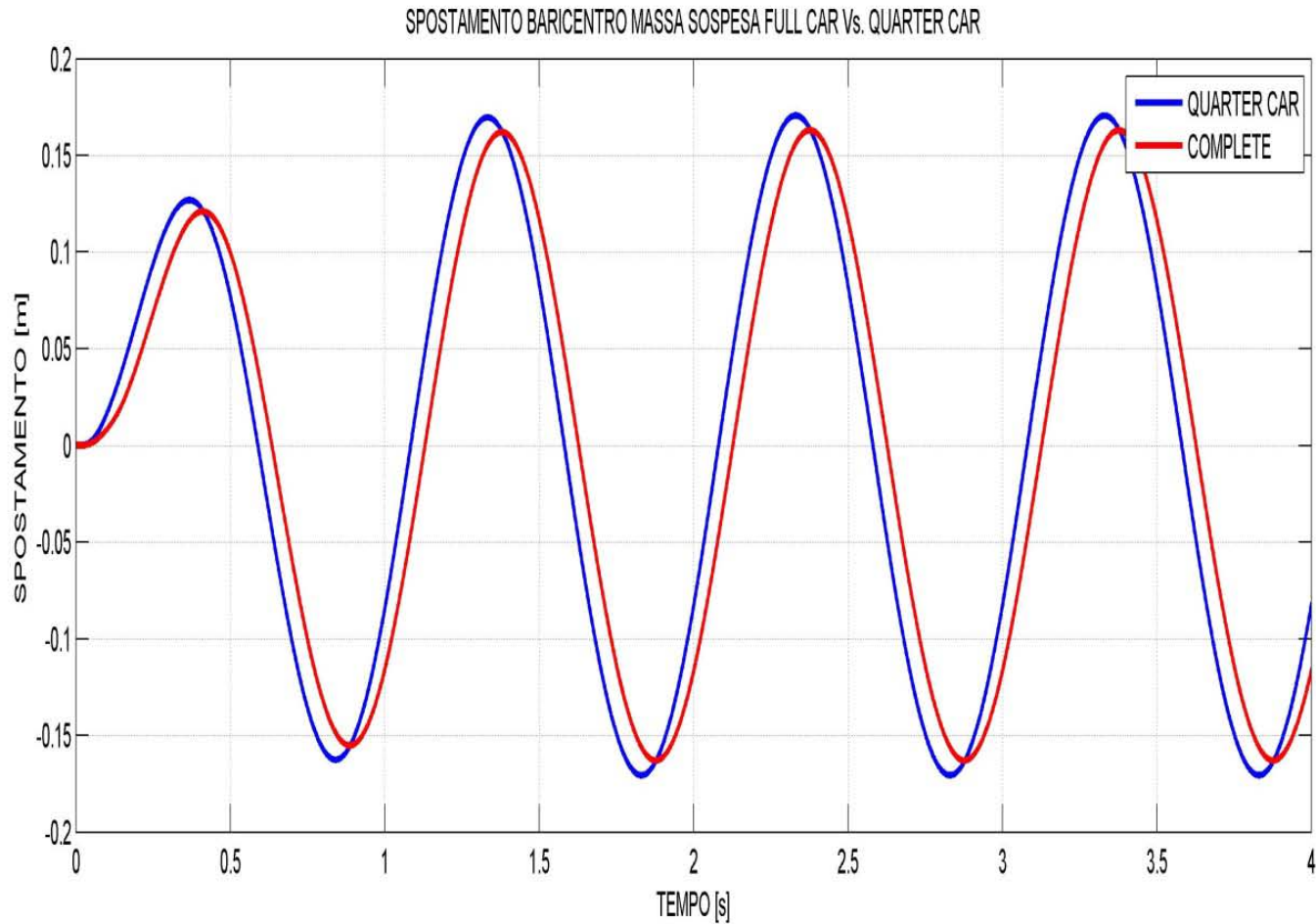


# QUARTER-CAR Vs. FULL-CAR





# MODELLO QUARTER-CAR Vs. FULL-CAR





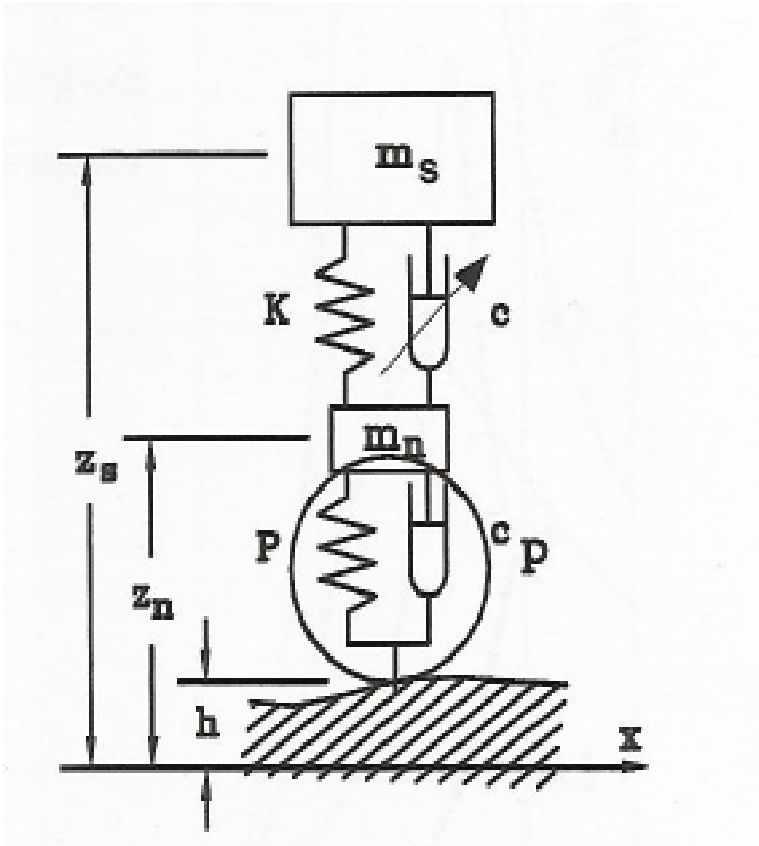
# MODELLO QUARTER-CAR CON CONTROLLO ATTIVO

## INGRESSI

1. Sinusoide ad 1 hz e amp= 0.1m
2. Sinusoide a 10 hz e amp=0.1 m
3. Scalino di 0.1 m

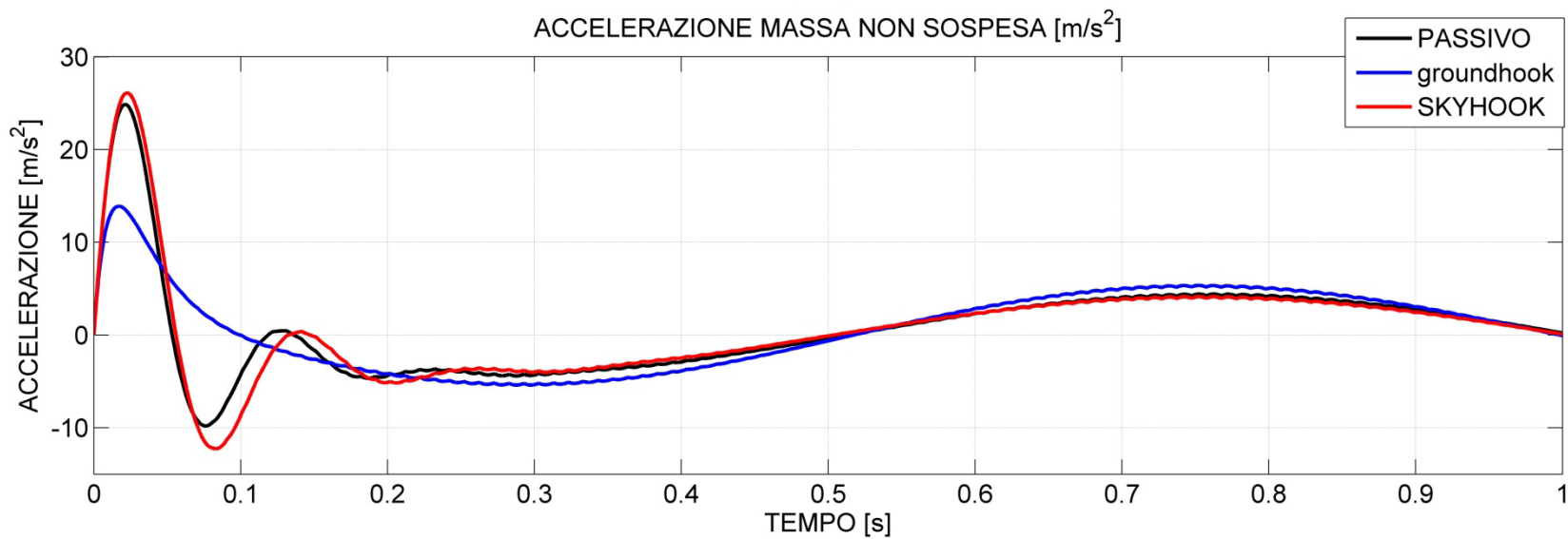
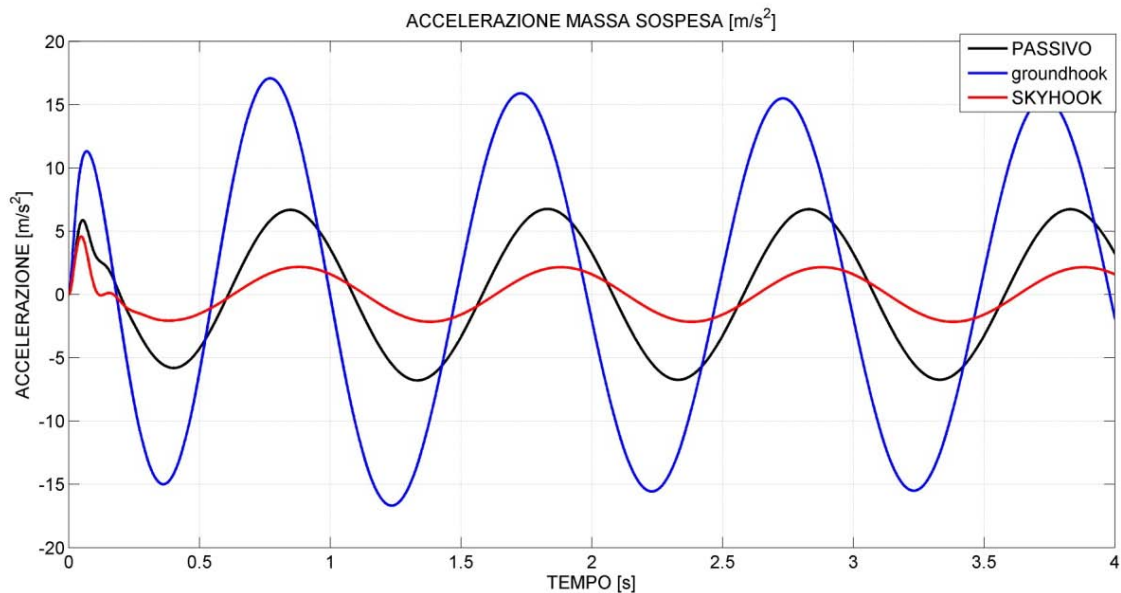
## OUTPUT

1. Accelerazione massa sospesa
2. Accelerazione massa non sospesa

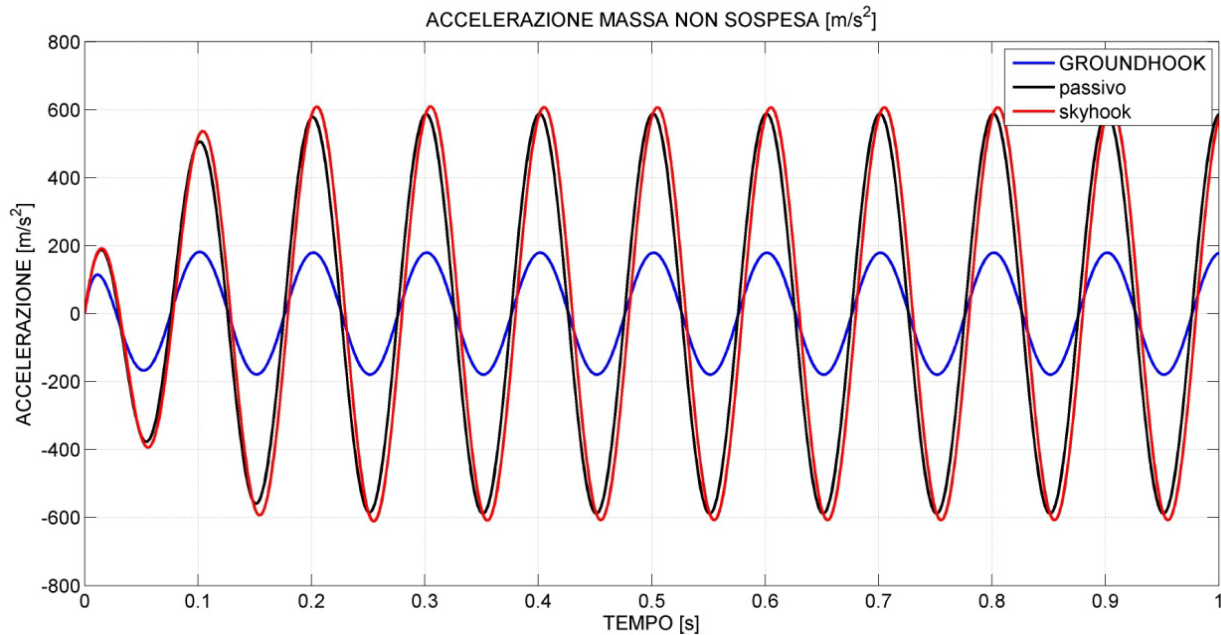
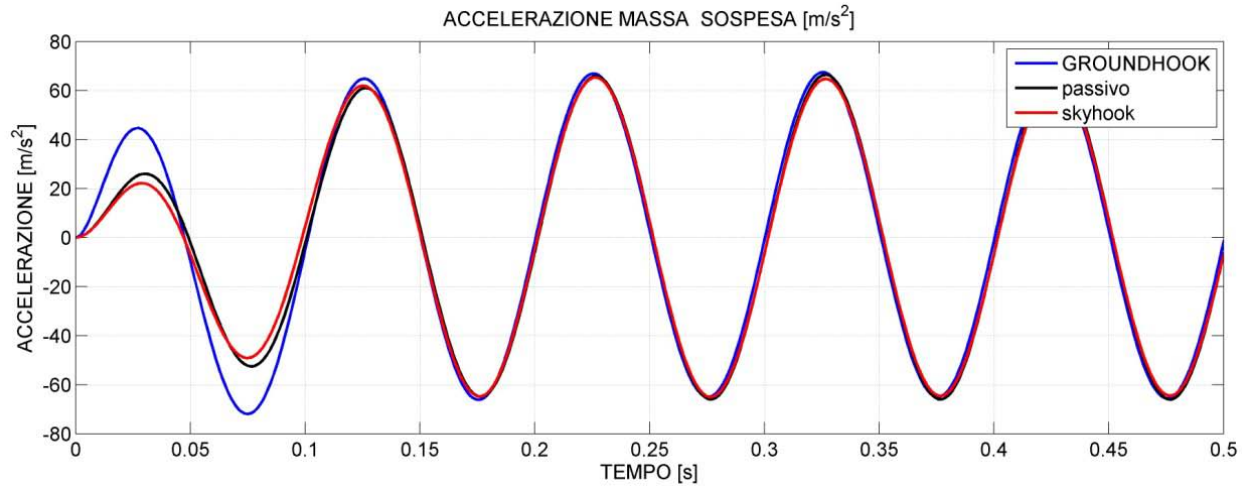




# INGRESSO SINUSOIDALE 1 Hz

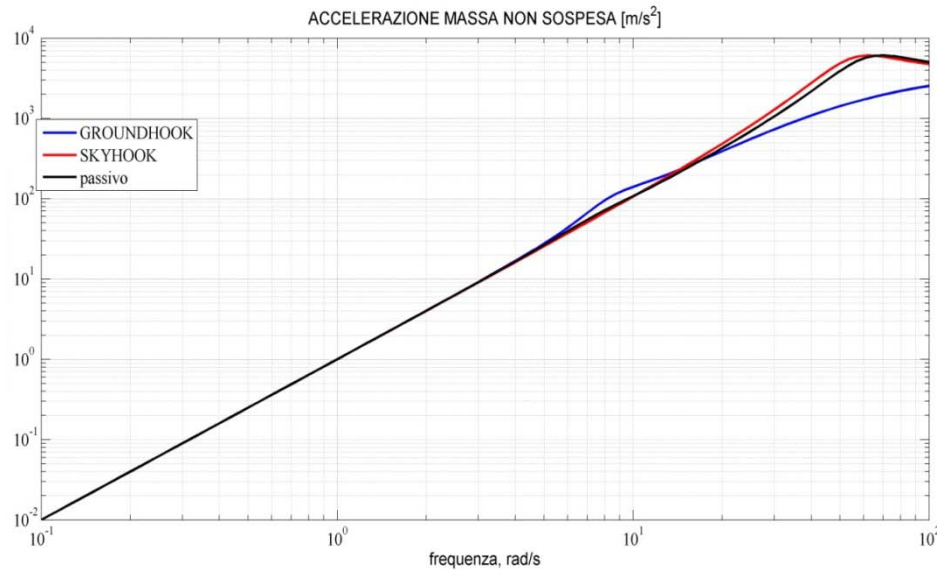
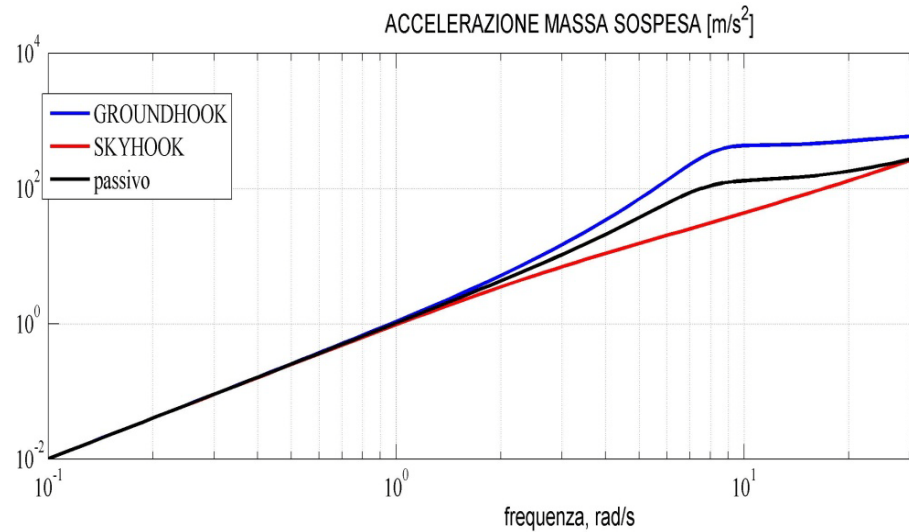


# INGRESSO SINUSOIDALE 10 Hz





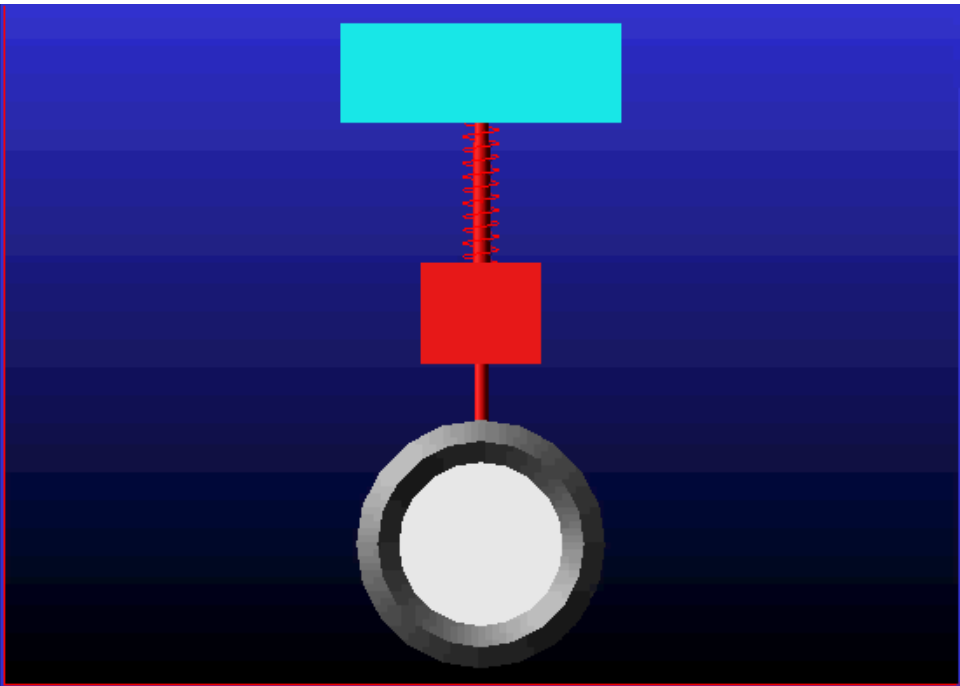
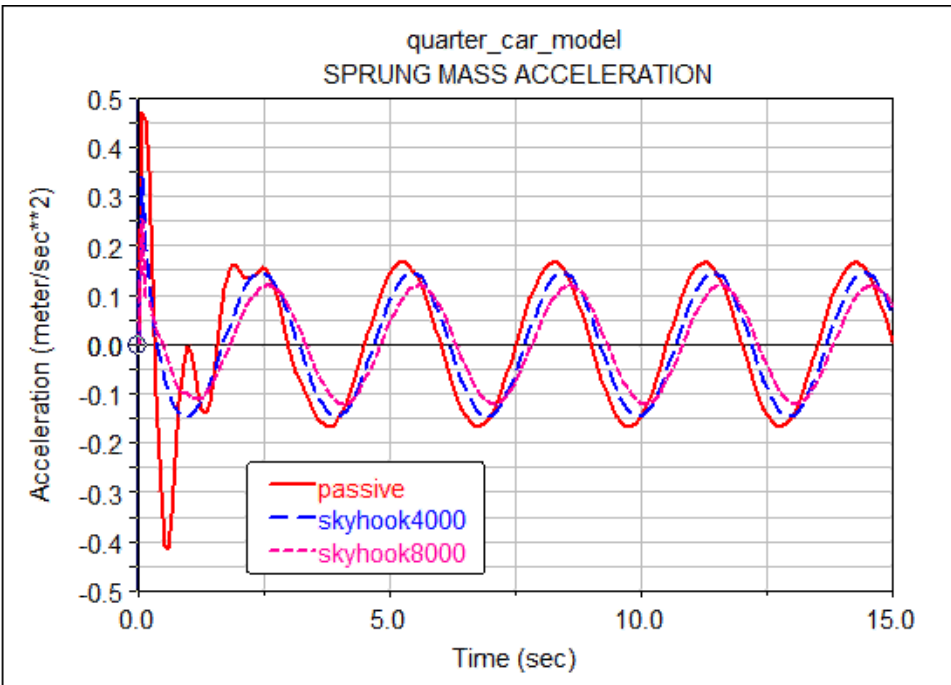
# ANALISI NEL DOMINIO DELLA FREQUENZA





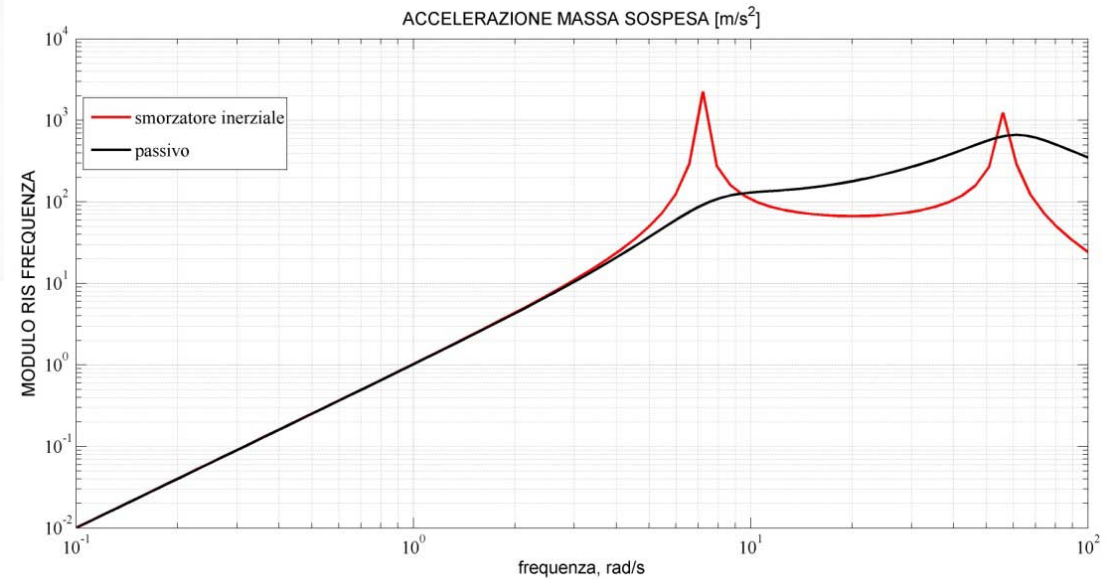
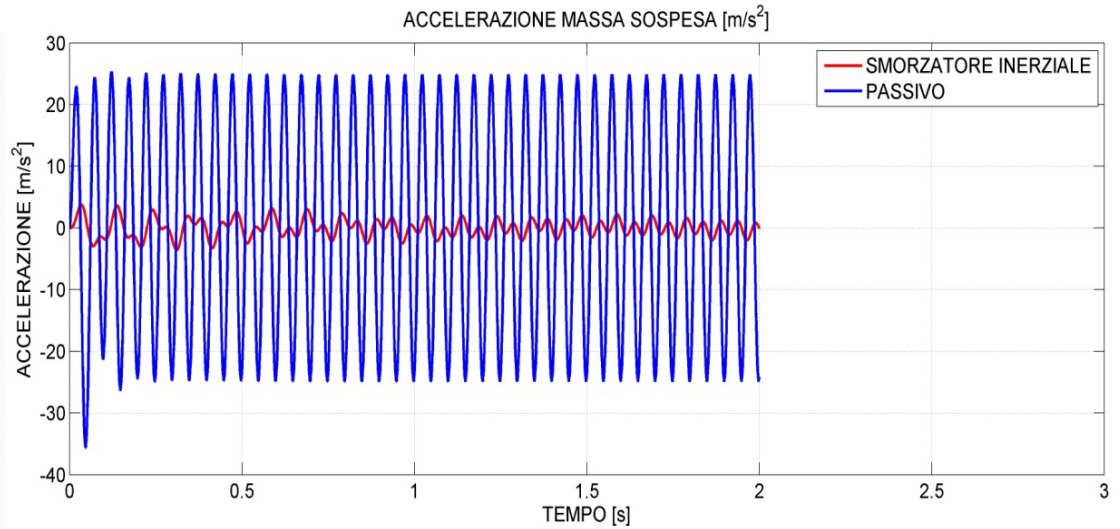
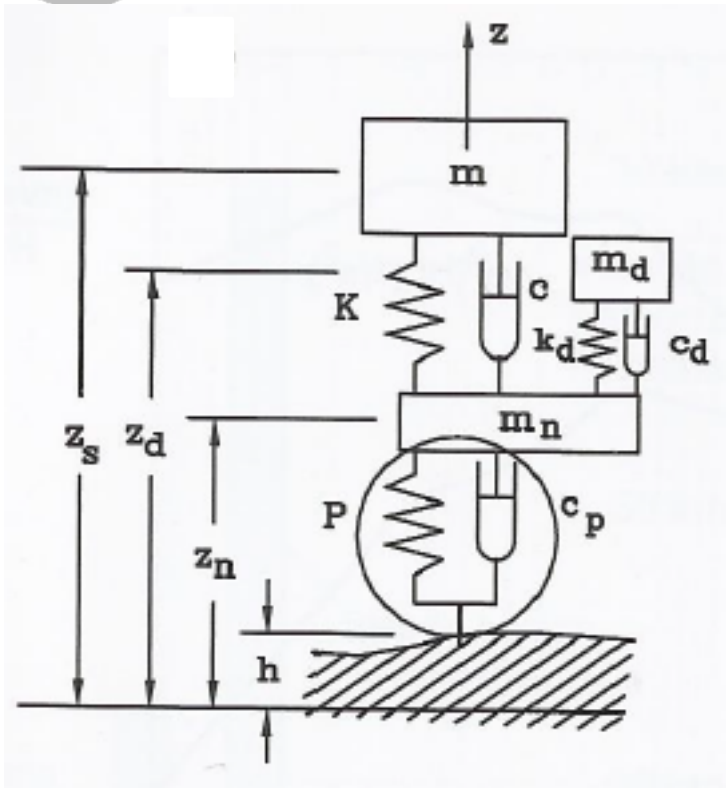


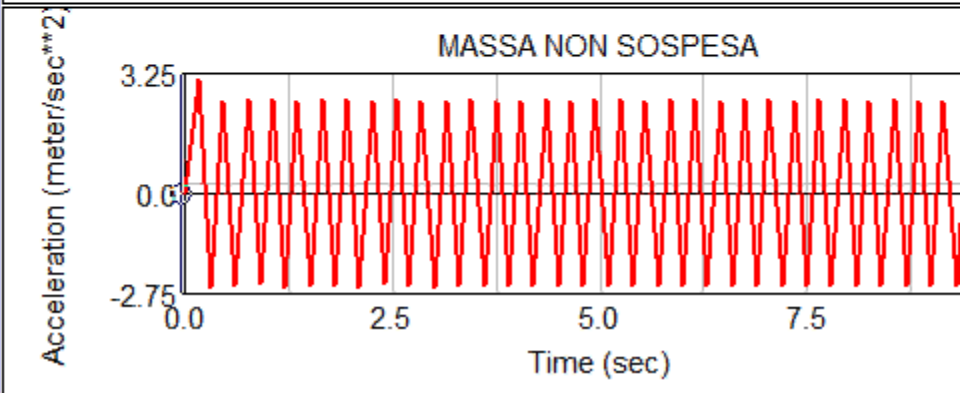
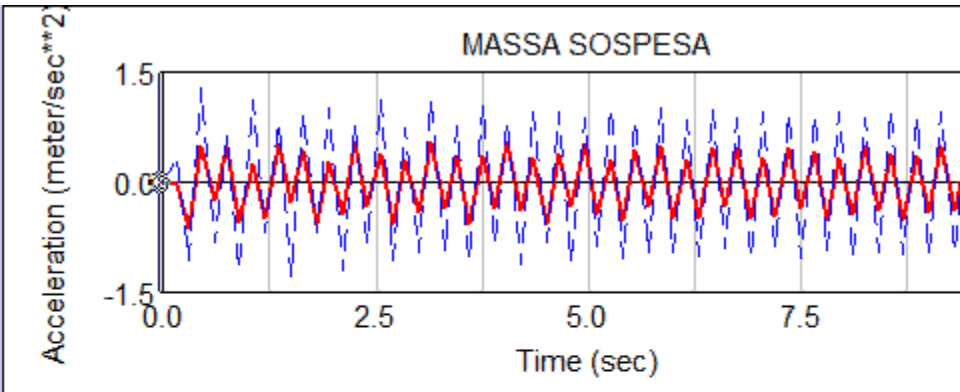
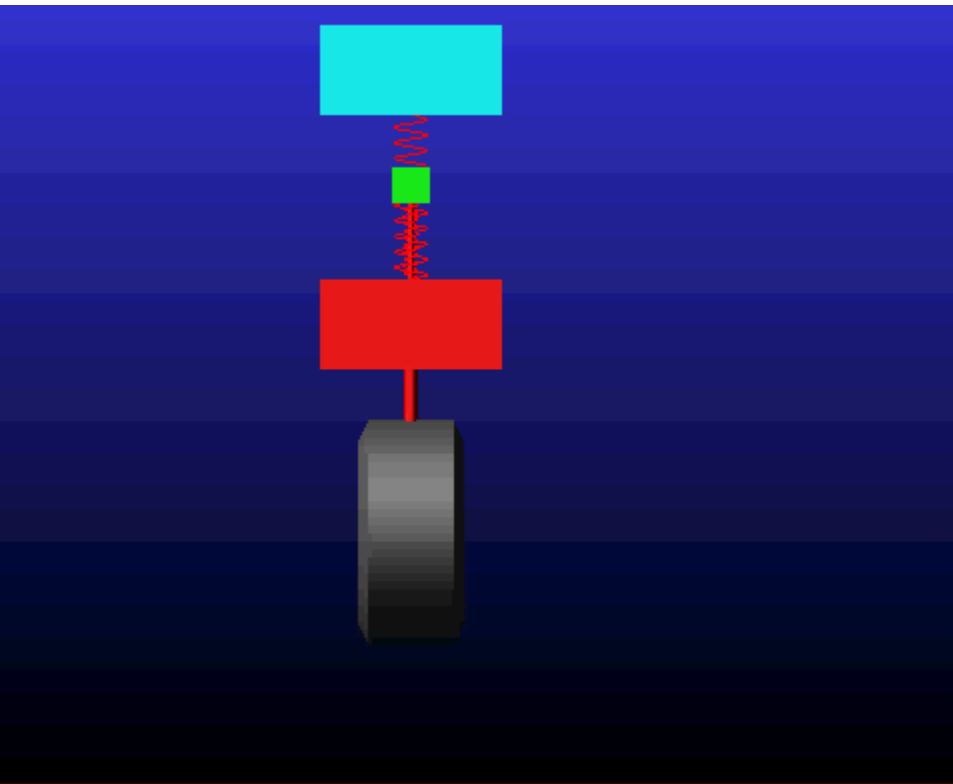
# SIMULAZIONE IN AMBIENTE MSC ADAMS





# MASS-DAMPER: CONVIENE?

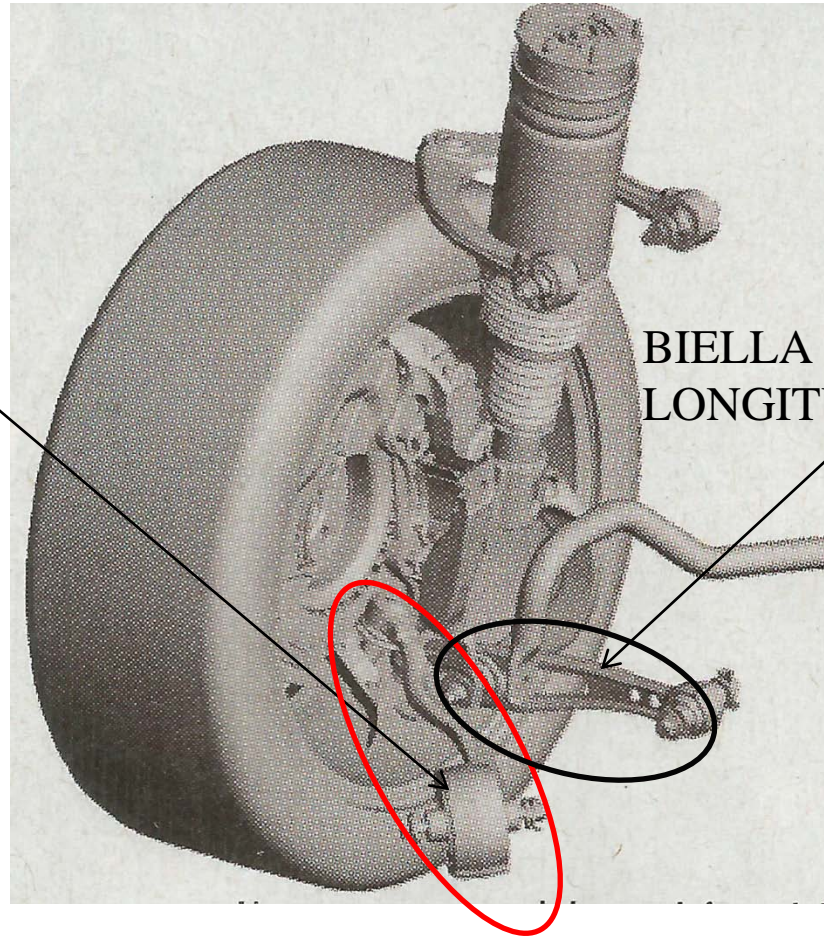




# SVILUPPI FUTURI: SOSPENSIONE REALE

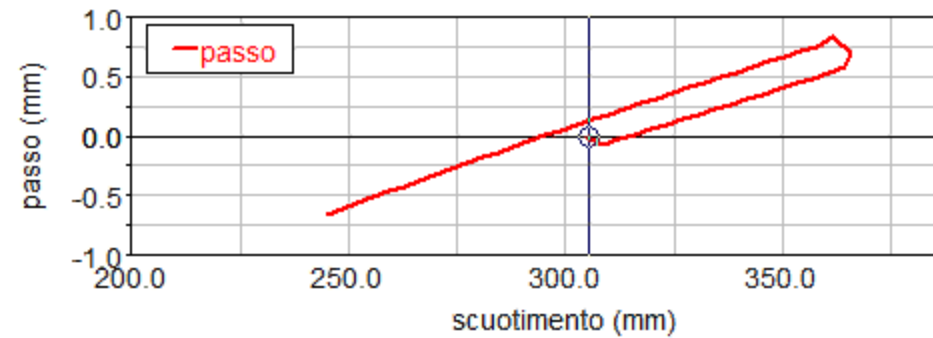
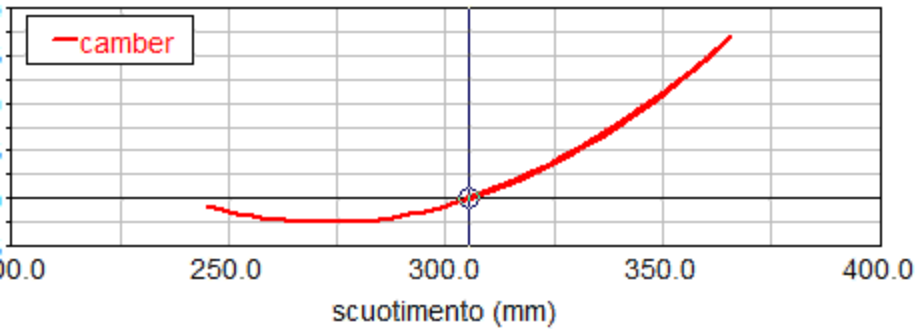
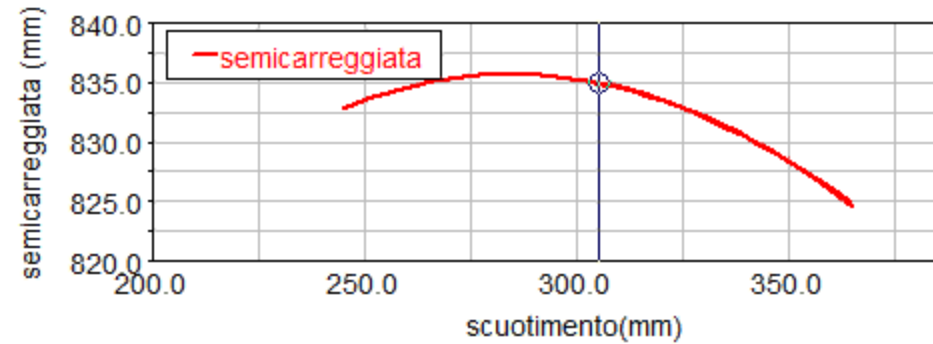
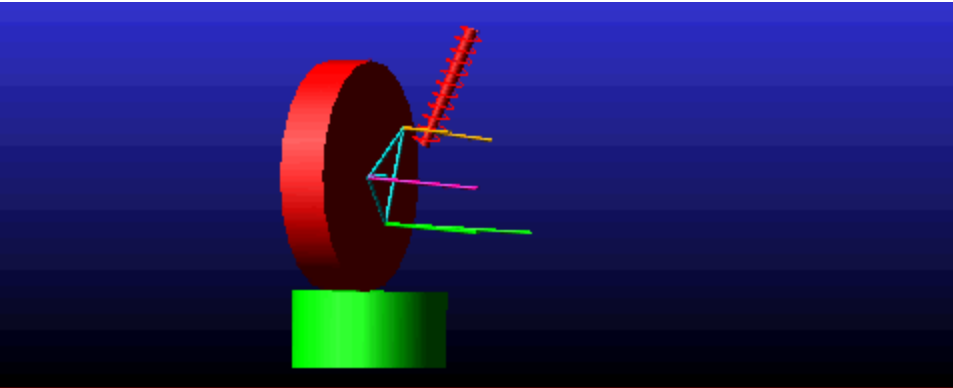
BIELLA  
TRASVERSALE

BIELLA  
LONGITUDINALE





# SVILUPPI FUTURI: SOSPENSIONE REALE





# CONCLUSIONI

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- **Utilizzare modello quarter-car in luogo di altri più complessi è una valida alternativa a patto di sapere bene cosa si vuole analizzare**
- **La sospensione attiva con strategia skyhook e groundhook consente un netto miglioramento delle prestazioni del sistema sospensivo**
- **Lo smorzatore inerziale posto su massa sospesa è una valida alternativa all' utilizzo di sistemi più complessi e costosi di ammortizzamento**